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AN INVESTIGATION OF AIRCRAFT HEATERS

VII - THERMAL RADIATION FROM ATHERMANOUS EXHAUST GASES

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ADVANCE RESTRICTED REPORT

AN INVESTIGATION OF ALRCRAFT HEATERS

VII - THERMAL RADIATION FROM ATHERMANOUS* EXHAUST GASES

By R. C. Martinelli, E. H. Morrin, and L. M. K. Boelter

SUMLEY

Accountions and necessary data for the calculation of the gaseous radiation from water vapor and carbon dioxide in an exhaust gas heat exchanger are presented. A typical calculation is included.

HTRODUCTION

In exhaust gases which contain appreciable quantities of carbon dioxide and water vapor, the theorem radiation from those constituents of the gas to the surroundings is not always negligible.

Hibtel and Eghert have compiled the latest data which are useful for the calculation of this radiant fraction. Under conditions normally found in amplane heat exchangers, the most recent corrections proposed by these authors (references 2 and 3) do not greatly change their former suggestions (reference 1).

In this report the emission and absorption of radiant energy is calculated for water value and carbon didwide, and a correction is made to account for the overlapping emission and absorption frequencies of these two gases.

Other constituents of exhaust gases, such as carbon particles, caono, carbon monexide, and organic compounds, also emit and absorb energy, but usually their concentrations are small and little is known about their effects. The radiation related to the luminosity of the gases, however, can be calculated. Diatonic non-polar molecules, such as Og, Hg, and Mg, do not emit or absorb radiant energy at wavelengths important to radiant heat transfer

*Capuble of emitting and absorbing radiant energy.

- ag over-all absorptance factor of gas
- CCO2 correction factor for effect of total pressure on CO2 radiation evaluated from figure 2
- $c_{\rm H_2O}$ correction factor for effect of total pressure on $\rm H_2O$ radiation evaluated from figure 4
- K correction factor due to presence of both ${\rm CO_2}$ and ${\rm H_2O}$ evaluated from figure 5
- L mean beam length which is equal to 0.9 times the diameter for infinite cylinder radiating to walls
- P_{CO_2} partial pressure of CO_2 atmospheres
- $P_{\rm H_2O}$ partial pressure of $\rm H_2O$ atmospheres
- ^CR Rankine = ^CF + 460
- Tg absolute temperature of gas, OR
- $T_{\rm S}$ absolute temperature of surroundings, ${}^{\rm O}{\rm R}$
- $^{\varepsilon}(\text{CO}_2,\,T_g,\,P_{\text{CO}_2}\,L)$ emissivity of CO2 evaluated from figure 1 at T_g and $P_{\text{CO}_2}\,L$
- $\begin{array}{c} \text{(CO$_2$, $T_{_{\rm S}}$, $P_{{\rm CO}}$_2$ L $\frac{T_{_{\rm S}}}{T_{_{\rm S}}}$) emissivity of ${\rm CO}_2$ evaluated from figure 1 at $T_{_{\rm S}}$ and $P_{{\rm CO}}$_2$ L $\frac{T_{_{\rm S}}}{T_{_{\rm S}}}$ } \end{array}$
- $^{\rm c}({\rm H_2O},~{\rm T_g},~{\rm P_{H_2O}}~{\rm L})$ emissivity of ${\rm H_2O}$ evaluated from figure 3 at T $_{\rm g}$ and ${\rm P_{H_2O}}~{\rm L}$

$$\begin{array}{c} \varepsilon \\ \text{MgO, T_S, P_{L_QO} L $\frac{T_S}{T_g}$} \end{array} \begin{array}{c} \text{cmissivity of $H_{2}O$ evaluated from} \\ \text{figure 3 at T_S and $P_{E_{2}O}$ L $\frac{T_S}{T_g}$} \end{array}$$

eg - cyer-all emissivity ractor of gas

 $\mathbf{c}_{\mathbf{g}}^{-}=\mathbf{e}(\mathbf{d}\mathbf{s}\mathbf{s}), \mathbf{f}^{+}, \ \mathbf{of} \ \mathbf{surrounding} \ \mathbf{surface}$

DISCUSSION

To ill strate the use of the charts and equations for the calculation of the vadiant energy interchange between gases containing water raper and curbon distribe and the surrounding softace, the following example is worked out:

Exhaust g s $\tau_{\rm g} = 1300^{\circ} \, \mathrm{T}, \, T_{\rm g} = 1760^{\circ} \, \mathrm{R}$

Exhaust pipe or heater walls, $\tau_{\rm g} = 900^{\circ}$ F, $T_{\rm g} = 1550^{\circ}$ R

I.D. of etherat pipe = 0.5 ft (0 in.)

Total presoure = 0.8 oras

Volume percent $L_2\theta$ raper = 15 percent = 0.075 atm

Volume percent $CO_2 = 15$ percent = 0.075 atm

The radiant energy interchange may be calculated from:

$$\left(\frac{1}{A}\right)_{H_2O} + co_2 = 0.1738 \quad \epsilon_s \quad \left[\epsilon_g \left(\frac{T_g}{100}\right)^4 - a_g \left(\frac{T_s}{100}\right)^4\right] \quad (1)$$

Where

$$\epsilon_{\text{g}} = \begin{bmatrix} c_{\text{CO}_2} & \epsilon_{\text{(CO}_2, \mathcal{I}_{\underline{c}}, \mathcal{P}_{\text{CO}_2}, \mathcal{I}_{\underline{c}}) + c_{\underline{E}_2} c_{\underline{c}} & \epsilon_{\text{(F}_2, c_2, \mathcal{I}_{\underline{c}}, \mathcal{P}_{\underline{c}}, c_{\underline{c}})} \end{bmatrix} (2)$$

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$$a_{3} = \left[{}^{c}\text{CO}_{2} \right] \left({}^{c}\text{CO}_{3}, T_{8}, P_{\text{CO}_{2}} \perp \frac{T_{8}}{T_{8}} \right) \left(\frac{T_{5}}{T_{8}} \right)^{0.65}$$

$$+ {}^{c}\text{H}_{2}\text{O} \left({}^{c}\text{H}_{2}\text{O}, T_{8}, P_{\text{T}_{2}\text{O}} \perp \frac{T_{3}}{T_{6}} \right) \left(\frac{T_{5}}{T_{8}} \right)^{0.45} - K$$
(5)

Frample is evaluated for altitude at which total pressure is 0.5 atm.

Determining numerical magnitudes

$$L = 0.9 \times 0.5 \text{ ft} = 0.45 \text{ ft}$$

$$P_{CO_{\circ}} L = P_{H_2O} L = 0.075 \times 0.45 = 0.0338 \text{ ft atm}$$

$$P_{CC_2} = \frac{T_5}{T_g} = P_{H_2O} \times E_{\frac{T_5}{T_g}} = 0.075 \times 0.45 \times \frac{1360}{1760} = 0.0261$$

 $\epsilon_{\rm g}$ = 0.79 (average value for oxidized steel)

$$\epsilon_{(CO_2, T_3, P_{CO_2} L)} = 0.052$$
 (from fig. 1)

$$\left(\text{CO}_{2}, \text{ T}_{s}, \text{ F}_{\text{CO}_{2}} \text{ L} \frac{\text{T}_{s}}{\text{T}_{s}}\right) = 0.047 \text{ (from fig. 1)}$$

$$\epsilon(\mathrm{H}_2\mathrm{O},\,\mathrm{T}_\mathrm{g},\,\mathrm{P}_{\mathrm{H}_2\mathrm{O}}\,\mathrm{L})$$
 = 0.028 (from fig. 3)

$$\epsilon \left(T_{2}O, T_{8}, P_{H_{2}O} L \frac{T_{8}}{T_{8}} \right) = 0.030 \text{ (from fig. 3)}$$

$$C_{CO_2} = 0.78$$
 (from fig. 2)(for total pressure = 0.5 atm)

$$C_{\underline{E}_2O} = 0.88$$
 (from fig. 4)(for total pressure = 0.5 atm)

$$K = 0.00$$
 (from fig. 5)

Therefore, from equations (3) and (3):

$$\epsilon_3 = 0.78 \times 0.052 + 0.88 \times 0.028 - 0.00 = 0.0651$$

$$a_{6} = \left[0.78 \times 0.047 \times \left(\frac{1760}{1300}\right)^{0.35} + 0.88 \times 0.030 \times \left(\frac{1760}{1360}\right)^{0.45} - 0.00\right]$$
$$= \left[0.0433 + 0.0296 - 0.00\right] = 0.0729$$

Then from equation (1)

$$\left(\frac{q}{\Lambda}\right)_{\text{CO}_2 + \text{H}_2\text{O}} = 0.1728 \times 0.79 \left[0.0651 \left(\frac{1760}{100}\right)^4 - 0.0729 \left(\frac{1360}{100}\right)^4\right]$$
$$= 0.137 \left[3250 - 2500\right] = 0.137 \times 3750$$
$$= 512 \text{ Btu/nr ft}^2$$

If this radiant energy interchange took place in an exhaust heater which was 0.5 foot in diameter and 3 feet in length, then

$$q(CO_2 + H_2O) = 512 (\pi \times 0.5 \text{ ft} \times 3 \text{ ft})$$

= 2410 Btu/hr

If the thermal output were 50,000 Btu/hr total exchange, then the fraction due to athermanous gas radiation would be 4.8 percent of the total.

In the experiments conducted in the University of California Mechanical Engineering Laboratory on straight and finned double tube heat exchangers (references 4 and 5), the athermancus gas radiation was less than 1 percent of the total transfer, because of the low concentrations of ${\rm CO_2}$ and ${\rm H_2O}$ vapor.

CONCLUSIONS

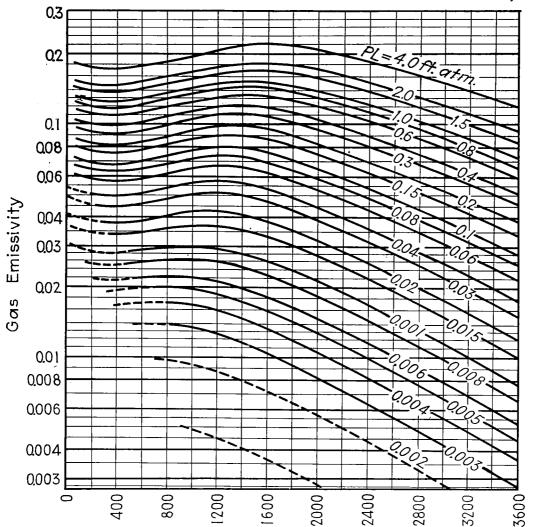
The radiant interchange taking place in an exhaust gas heat exchanger due to the emission and absorption characteristics of water vapor and carbon dioxide may be calculated, employing gaseous radiation data of Hottel. There are not sufficient data to account accurately for the radiant energy interchange from other gases, such as carbon monoxide and hydrocarbons.

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RUFFERENCES

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- 2. McAdams, W. H.: Heat Transmission. McGraw-Hill Book Co., Inc., New York, N. Y., 1942, pp. 64, 73.
- 3. Hottel, H. C., and Eghert, R. B.: Radiant Heat Transmission from Water Vapor. Trans., Am. Inst. of Chem. Eng., vol. 38, no. 3, June 25, 1942, p. 531.
- 4. Martinelli, R. C., Weinberg, E. B., Morrin, E. H., and Boelter, L. M. K.: An Investigation of Aircraft Heaters. III Measured and Predicted Performance of Double Tube Heat Exchangers. NACA ARR, Oct. 1942.
- Martinelli, R. C., Weinberg, E. B., Morrin, E. H., and Boelter, L. M. K.: An Investigation of Aircraft Heaters. IV - Measured and Predicted Porformance of Longitudinally Finned Tubes. NACA APR, Cct. 1942.
- NOTE: The figures of this report have been reproduced from "Radiant Heat Transmission from Water Vapor," by H. C. Hottel and R. B. Egbert, appearing in vol. 38, no. 3 of the Transactions of the American Institute of Chemical Engineers, with the permission of the authors. The ordinates of figures 2, 4, and 5 have been changed as indicated below:
 - Fig. 2, crdinate is renamed CCO,
 - Fig. 4, ordinate is renamed $C_{\rm H_2O}$
 - Fig. 5, ordinate is renamed K

The coordinates of figures 1 and 3 remain unchanged.



(This is figure 22 of reference 3)

Temperature ° F

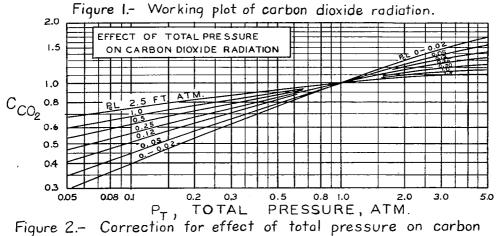
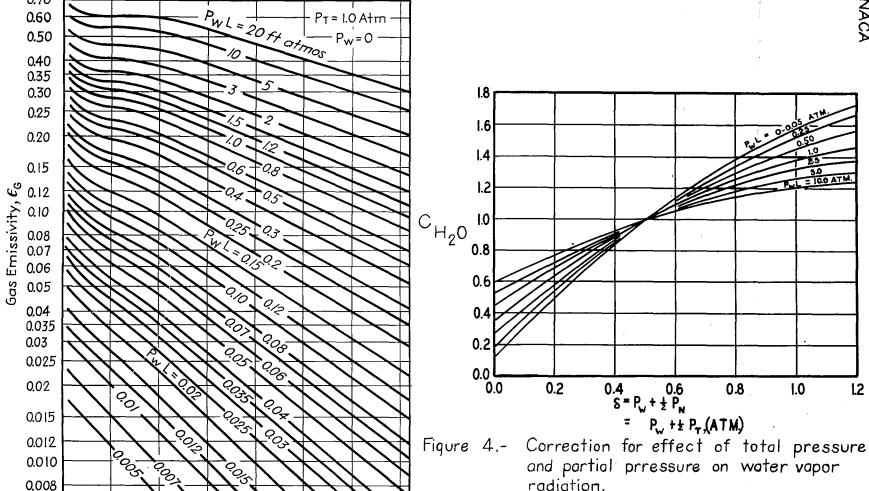


Figure 2.dioxide radiation.

(This is figure 21 of reference 3. Ordinate scole renamed)







radiation. 0.007 (This is figure 20 of reference 3. Ordinate scale renamed)

500 1000 1500 2000 2500 3000 3500 Temperature, Deg F Final working plot of water vapor emissivity, Figure 3.for $P_w = 0$. (This is figure 18 of reference 3)

0.70

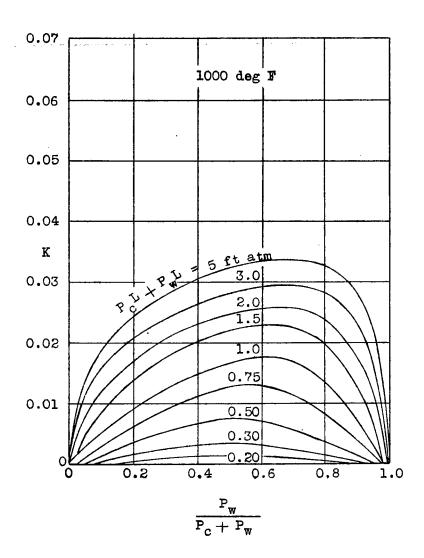


Figure 5.- Correction for superimposed radiation from mixtures of carbon dioxide and water vapor.

Ordinate scale renamed.

This figure taken from figure 19 reference 3.

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